

# Notes on Nonlinear Control

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## 1 Second-order Systems

### 1.1 Basics

Every second order system can be brought in the state-space form

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} f_1(x_1, x_2, t) \\ f_2(x_1, x_2, t) \end{bmatrix}.$$

For second-order systems, we are mostly interested in the phase plane  $(x_2, x_1)$ . The angle of the trajectory is given by

$$\theta(f(\mathbf{x})) = \tan^{-1} \frac{\dot{x}_2}{\dot{x}_1} = \tan^{-1} \frac{f_2(x_1, x_2, t)}{f_1(x_1, x_2, t)},$$

at a point  $\vec{x}$ .

### 1.2 Linear Systems

Every linear system  $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x}$  can be transformed into an equivalent, canonical linear system by  $\mathbf{z} = \mathbf{Q}^{-1}\mathbf{x}$ , where the qualitative behavior of both is equivalent, and the one of  $\mathbf{z}$  is denoted by

$$\dot{\mathbf{z}} = \mathbf{Q}^{-1}\mathbf{A}\mathbf{Q}\mathbf{z} = \mathbf{\Lambda}\mathbf{z}.$$

The matrix  $\mathbf{\Lambda}$  is determined by the Eigenvalues. The Eigenvalues can be computed by

$$\lambda_{1/2} = \frac{T}{2} \pm \frac{T}{2} \sqrt{T^2 - 4\Delta},$$

where  $T = \text{Tr } \mathbf{A}$ , and  $\Delta = \det \mathbf{A}$ . This yields several cases as shown in Table 1. The local coordinate system is then given by the

$$\mathbf{x} = \mathbf{q}_1 z_1 + \mathbf{q}_2 z_2,$$

where  $\mathbf{Q} = [\mathbf{q}_1, \mathbf{q}_2]$ . Hence you can just draw each coordinate system by by setting one of the  $z_i = 0$ .

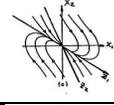
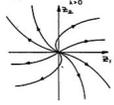
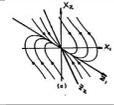
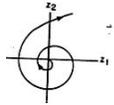
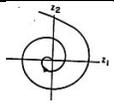
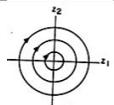
Eigenvalues	Type	Picture
$\lambda_i > 0$ , and $\lambda_i \in \mathbb{R}$	Stable node	
$\lambda_i < 0$ , and $\lambda_i \in \mathbb{R}$	Unstable node	
$\lambda_1 \cdot \lambda_2 < 0$ , and $\lambda_i \in \mathbb{R}$	Saddle point	
$\lambda_i = \alpha \pm j\beta$ , and $\lambda_i \in \mathbb{C}$	Unstable focus	
$\lambda_i = \alpha \pm j\beta$ , and $\lambda_i \in \mathbb{C}$	Stable focus	
$\lambda_i = \pm j\beta_i$ , and $\lambda_i \in \mathbb{C}$	Center	

Table 1: A table showing all main behaviors of second order systems

### 1.3 Second-order linear differential equation

The general second order linear differential equation can be given as

$$\ddot{y} + 2\xi\omega\dot{y} + \omega^2 y = 0,$$

and it has the Eigenvalues  $\lambda_{1/2} = -\xi\omega \pm \omega\sqrt{\xi^2 - 2}$ . This implies that it can be a stable focus ( $0 < \xi < 1$ ), a stable node ( $\xi > 1$ ), a center ( $\xi = 0$ ), an unstable focus ( $-1 < \xi < 0$ ), or an unstable node ( $\xi < -1$ ).

### 1.4 Analysis by Linearization

We can linearize the system around an equilibrium point  $\mathbf{x}_{eq}$  for which  $f(\mathbf{x}_{eq}) = 0$ . When defining  $\Delta\mathbf{x} = \mathbf{x} - \mathbf{x}_{eq}$ , we can linearize the local solution and get

$$\begin{aligned} \Delta\dot{x}_1 &= \frac{\partial f_1}{\partial x_1} \Delta x_1 + \frac{\partial f_1}{\partial x_2} \Delta x_2 + \epsilon_1, \\ \Delta\dot{x}_2 &= \frac{\partial f_2}{\partial x_1} \Delta x_1 + \frac{\partial f_2}{\partial x_2} \Delta x_2 + \epsilon_2, \end{aligned}$$

where  $\epsilon \in O(\Delta x^2)$ . For small  $\Delta x_i$  the linearized system yields similar behavior as the nonlinear one except for the center.

## 1.5 Drawing Phase Planes

This gives a recipe for drawing phase planes:

1. Horizontal axis is denoted by  $x_1$ , vertical axis by  $x_2$ .
2. Equilibrium Points
  - (a) Determine equilibrium points by  $f(\mathbf{x}_{\text{eq}}) = 0$ .
  - (b) Linearize around equilibrium points by  $A_{ij} = \partial f_i / \partial x_j$ .
  - (c) Determine equilibrium point type from Eigenvalues by

$$\lambda_{1/2} = \frac{T}{2} \pm \frac{T}{2} \sqrt{T^2 - 4\Delta},$$

where  $T = \text{Tr } \mathbf{A}$ , and  $\Delta = \det \mathbf{A}$

- (d) Determine local coordinate system or Eigenvectors from Table 1.
  - (e) Draw local trajectories using the example.
3. Determine Isoclines
  - (a) Vertical arrows up where  $\dot{x}_1 = 0$ ,  $\dot{x}_2 > 0$ .
  - (b) Vertical arrows down where  $\dot{x}_1 = 0$ ,  $\dot{x}_2 < 0$ .
  - (c) Horizontal arrows to the right where  $\dot{x}_2 = 0$ ,  $\dot{x}_1 > 0$ .
  - (d) Horizontal arrows to the left where  $\dot{x}_2 = 0$ ,  $\dot{x}_1 < 0$ .
4. Use symmetry if possible.
5. Draw Trajectories which do not intersect.

## 1.6 Special case: $\ddot{y} = g(\dot{y}, y)$

The system  $\ddot{y} = g(\dot{y}, y)$  becomes  $\dot{x}_1 = \dot{y} = x_2$ ,  $\dot{x}_2 = g(\dot{y}, y) = g(x_2, x_1)$  in state-space form. It has the following properties:

- All equilibrium points intersect with the horizontal axis.
- All trajectories have vertical slope at the horizontal axis.
- If  $|g(\dot{y}, y)|$  is bounded, the vertical slope can only occur on the horizontal axis.

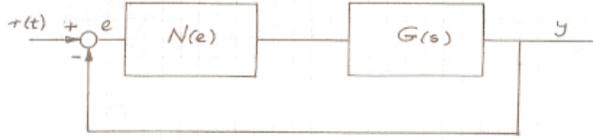


Figure 1: Separation of the linear and nonlinear block.

## 2 Piecewise linear Switching Systems

### 2.1 Response to a Step Impulse

The response to a step input can be analyzed as follows:

1. Separate the system into a nonlinear system  $N(e, \dot{e})$  and a linear system  $F(s)$  as in Figure 1.
2. Transform the  $F(s) = Y(s)/U(s)$  into time-domain where  $f(y, \dot{y}, \ddot{y}) + u = 0$ .
3. The step input  $r(t) = A \cdot 1(t)$ , yields  $e = r - y = A - y$ , which implies

$$y = A - e, \dot{y} = -\dot{e}, \ddot{y} = -\ddot{e},$$

and therefore

$$f(A - e, -\dot{e}, -\ddot{e}) + N(e) = 0.$$

4. Separate all linear regions of  $N(e)$  by decision borders, and rename the  $x_1 = e$ , and  $x_2 = \dot{e}$ .
5. Analyze the different piece-wise linear regions. Use separation of variable for

$$\frac{dx_2}{dx_1} = h(x_1, x_2),$$

if possible. Three practical cases: (i) Lines  $h(x_1, x_2) = const$ , (ii) Parabola  $h(x_1, x_2) = B/x_2$ .

### 2.2 Response to a Ramp Impulse

The response to a step input can be analyzed as follows:

1. Separate the system into a nonlinear system  $N(e, \dot{e})$  and a linear system  $F(s)$  as in Figure 1.
2. Transform the  $F(s) = Y(s)/U(s)$  into time-domain where  $f(y, \dot{y}, \ddot{y}) + u = 0$ .

3. The step input  $r(t) = At$ , yields  $e = At - y$ , which implies

$$y = At - e, \dot{y} = A - \dot{e}, \ddot{y} = -\ddot{e},$$

and therefore

$$f(At - e, A - \dot{e}, -\ddot{e}) + N(e) = 0.$$

4. Separate all linear regions of  $N(e)$  by decision borders, and rename the  $x_1 = e$ , and  $x_2 = \dot{e}$ .

5. Analyze the different piece-wise linear regions. Use separation of variable for

$$\frac{dx_2}{dx_1} = h(x_1, x_2),$$

if possible. Three practical cases: (i) Lines  $h(x_1, x_2) = \text{const}$ , (ii) Parabola  $h(x_1, x_2) = B/x_2$ .

## 3 Conservative Systems

### 3.1 Basics

A conservative system is given by

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ -f(x_1) \end{bmatrix},$$

which implies

$$\frac{dx_2}{dx_1} = -\frac{f(x_1)}{x_2} \Rightarrow \frac{x_2^2}{2} + \int_{x_{10}}^{x_1} f(x_1) dx_1 = T + V = E,$$

where  $T$  is kinetic energy,  $V$  is potential energy, and  $E$  the total energy determined by the initial conditions. Trajectories can be determined by  $x_2 = \pm \sqrt{2(E - V(x_1))}$ .

### 3.2 Interesting Notes

Interestingly, we have:

- The equilibrium points are at the extrema of the potential energy

$$f(x_1) = \frac{\partial V}{\partial x_1} = 0.$$

- Maxima of  $V(x_1)$  are stable equilibrium points.
- Minima of  $V(x_1)$  are instable equilibrium (saddle) points.

## 4 Describing Function Method

### 4.1 Basics

DFM is employed to determine self-oscillation.

1. Assume a time-invariant  $N(e)$ ,  $e = \hat{e} \sin \omega t$ .
2. Determine  $y(t)$  by Fourier series expansion while neglecting bias ( $\alpha_0$  or  $k = 0$ ), and higher order harmonics ( $k > 2$ ). The Fourier coefficients are

$$\alpha_1 = \int_{t_0}^{t_0+T} y(t) \cos(\omega t) dt,$$
$$\beta_1 = \int_{t_0}^{t_0+T} y(t) \sin(\omega t) dt.$$

3. This yields

$$y(t) = \alpha_1 \cos \omega t + \beta_1 \sin \omega t = (\beta_1 + \alpha_1 j) \sin \omega t = N_1 \sin \omega t,$$

where  $N_1 = \sqrt{\alpha_1^2 + \beta_1^2} e^{j\phi}$  with  $\phi = \tan^{-1}(\alpha_1/\beta_1)$ .

4. Determine the equivalent gain or describing function

$$\eta(\hat{e}, \omega) = \frac{1}{\hat{e}} \sqrt{\alpha_1^2 + \beta_1^2} e^{j\phi} = \frac{1}{\hat{e}} (\beta_1 + \alpha_1 j).$$

This yields the output oscillations

$$y(t) = \eta(\hat{e}, \omega) \hat{e} \sin \omega t.$$

### 4.2 Properties

We have proved the following properties in the lecture:

- A sinusoidal describing function for a memoryless nonlinearity is always real, i.e.,  $\alpha_1 = 0$ .
- If nonlinear characteristic  $N(e)$  is memoryless, and time-invariant, then the characteristic function is independent of the frequency  $\eta(\hat{e}, \omega) = \eta(\hat{e})$ .

### 4.3 Analysis of Limit Cycles with DFM

The input is assumed to be a pure sinusoid, and higher order harmonic effects are neglected. We call the linear system  $G(s)$ . This allows the following recipe:

1. Replace  $N(e)$  by  $\eta(\hat{e}, \omega)$ . Determine  $G(j\omega) = G(s)|_{s=j\omega}$ .

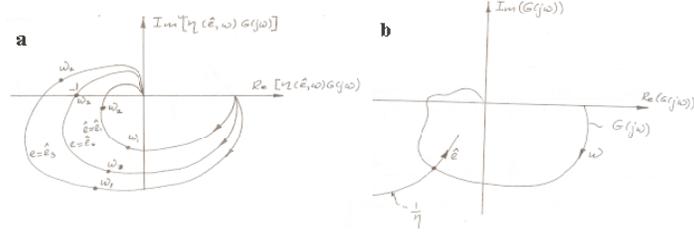


Figure 2: The Family of Nyquist Plots methods is described in (a), and the Nyquist plane of  $G(j\omega)$  is shown in (b).

2. A limit cycle exists if

$$G(j\omega) = -\frac{1}{\eta(\hat{e}, \omega)},$$

or equivalently

$$\begin{aligned}\eta_R G_R + \eta_I G_I &= -1, \\ \eta_R G_I + \eta_I G_R &= 0.\end{aligned}$$

3. This can be solved either by any of the following methods.

**Family of Nyquist Plots:** Plot imaginary component  $\text{Im}[G(j\omega)\eta(\hat{e}, \omega)]$  versus real component  $\text{Re}[G(j\omega)\eta(\hat{e}, \omega)]$  for different  $\hat{e}$ , and find the  $\hat{e}_{-1}$  for which  $G(j\omega)\eta(\hat{e}_{-1}, \omega) = -1$ . The  $\omega$  for which this is true is the frequency of the cycle. See Figure 2 (a).

**Nyquist plane of  $G(j\omega)$ :** Plot  $G(j\omega)$  as functions of  $\omega$ , and  $-1/\eta(\hat{e})$  as functions of  $\hat{e}$ . The  $\omega$ , and  $\hat{e}$  where they meet are frequency and amplitude of the cycle, respectively. See Figure 2 (b).

**Analytical method:** For memory-free nonlinearities, we can solve the equation  $\eta_R(\hat{e})G_I(j\omega) = 0$  for  $\omega = \omega_c$ . We substitute this and solve  $\eta_R(\hat{e})G_R(j\omega_c) = -1$  for  $\hat{e} = \hat{e}_c$ .

4. Determine closed loop stability as shown in Figure 3, or using

$$\max_{\hat{e}} \left( -\frac{1}{\eta(\hat{e})} \right) > G(j\omega_C).$$

## 5 Lyapunov Equilibrium Point Analysis

### 5.1 Definitions

**Equilibrium Point:** The point  $\mathbf{x}_0$  is an EQ iff  $f(\mathbf{x}_0, t) = 0$ .

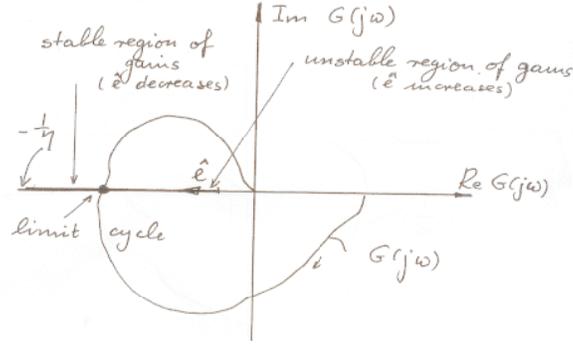


Figure 3: This figure shows the stability analysis for a system with a memory-free nonlinearity.

**Isolated EQP:**  $\mathbf{x}_0$  is an isolated EQ iff  $\mathbf{x}_0 \text{ EQ} \wedge \neg \exists \mathbf{x} \in B_\epsilon(\mathbf{x}_0) : f(\mathbf{x}, t) = 0$ .

**Lyapunov function:** Scalar function  $V(\mathbf{x}, t)$ , derivative is  $\dot{V}(\mathbf{x}, t) = \partial V / \partial t + (\partial V / \partial \mathbf{x})^T f(\mathbf{x}, t)$ .

**LDPF:**  $\forall \mathbf{x} \in B_\epsilon(\mathbf{x}_0) : V(\mathbf{x}, t) \geq W(\mathbf{x}) > 0$ , and  $V(\mathbf{0}, t) = W(\mathbf{0}) = 0$ .

**PDF:**  $\forall \mathbf{x} \in \mathbb{R}^n : V(\mathbf{x}, t) \geq W(\mathbf{x}) > 0$ , and  $V(\mathbf{0}, t) = W(\mathbf{0}) = 0$ .

**Radially unbounded:**  $\lim_{\|\mathbf{x}\| \rightarrow \infty} W(\mathbf{x}) \rightarrow \infty$ .

**Decrescent:**  $\forall \mathbf{x} \in \mathbb{R}^n, t \geq 0 : V(\mathbf{x}, t) \leq \hat{W}(\mathbf{x})$ .

## 5.2 Stability

**Stability:** An EQ is stable if

$$\forall \epsilon > 0 : \exists \delta(t_0, \epsilon) > 0 : \|\mathbf{x}(t_0)\| < \delta(t_0, \epsilon) \implies \|\mathbf{x}(t)\| < \epsilon, \forall t \geq t_0.$$

This is implied by:  $V$  lpdf, and  $\dot{V}(\mathbf{x}, t) \leq 0$ .

**Uniform Stability:** An EQ is uniformly stable if

$$\forall \epsilon > 0 : \exists \delta(\epsilon) > 0 : \|\mathbf{x}(t_0)\| < \delta(\epsilon) \implies \|\mathbf{x}(t)\| < \epsilon, \forall t \geq t_0.$$

This is implied by:  $V$  decrescent, lpdf, and  $\dot{V}(\mathbf{x}, t) \leq 0$ .

**Asymptotic stability:** An EQ is asymptotically stable iff  $\exists \delta > 0 : \|\mathbf{x}(t_0)\| < \delta \implies \lim_{t \rightarrow \infty} \|\mathbf{x}(t)\| = 0$ . This is implied by:  $V$  decrescent, lpdf, and  $-\dot{V}(\mathbf{x}, t)$  lpdf.

**Global Asymptotic Stability:**  $V$  decrescent, pdf, radially unbounded, and  $-\dot{V}(\mathbf{x}, t) \leq -\hat{W}(\mathbf{x})$ .

**Exponential Stability:** A EQ is exponentially stable iff  $\exists r, b, a > 0 : \|\mathbf{x}(t)\| \leq k \|\mathbf{x}(t_0)\| e^{-bt}, \forall t \geq t_0$ . This is given by:  $\forall \mathbf{x} \in B_\epsilon(\mathbf{x}_0) : a \|\mathbf{x}\|^p \leq V(\mathbf{x}, t) \leq b \|\mathbf{x}\|^p, \dot{V}(\mathbf{x}, t) \leq -c \|\mathbf{x}\|^p$ .

**Global Exponential Stability:**  $\forall \mathbf{x} \in \mathbb{R}^n : a \|\mathbf{x}\|^p \leq V(\mathbf{x}, t) \leq b \|\mathbf{x}\|^p, \dot{V}(\mathbf{x}, t) \leq -c \|\mathbf{x}\|^p$ .

### 5.3 Further Lyoponov Methods

**Instability theorem:** Choose a  $V$  so that  $+\dot{V}$  is lpdf, and  $V(\mathbf{0}, t) = 0$ . Show that  $V(\mathbf{x}, t) > 0$  for any point  $\mathbf{x}$  which is arbitrarily close to the origin.

**La-Salle Kravovski:**  $\mathbf{x} = \mathbf{0}$  is asymptotically stable if (i)  $V(\mathbf{x})$  lpdf, (ii)  $\Omega_e = \{\mathbf{x} | V(\mathbf{x}) \leq c\}$  is bounded, (iii)  $\dot{V}(\mathbf{x}) \leq 0$ , and (iv) the set  $S = \{\mathbf{x} \in \Omega_e | \dot{V}(\mathbf{x}) = 0\}$  contains no trajectories of the system (e.g.,  $V(0, x_2) = 0$ , and  $x_1 = 0 \implies \dot{x}_2 \neq 0$ ).

**Linear Time-Invariant Systems:** For a time-invariant system  $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x}$ , the origin is globally asymptotically stable iff equivalently (i)  $\forall i. \text{Re } \lambda_i(\mathbf{A}) < 0$ , or (ii) given a positive definite symmetric matrix  $\mathbf{Q}$ , we can solve

$$\mathbf{P}\mathbf{A} + \mathbf{A}^T\mathbf{P} = -\mathbf{Q}$$

for a unique, positive definite  $\mathbf{P}$ . (Note: Solving for  $\mathbf{P}$  given  $\mathbf{Q}$  is sufficient and necessary, solving for  $\mathbf{Q}$  given  $\mathbf{P}$  is *only* sufficient and *not* necessary).

**Linear Time-Variant Systems:** For the time-variant system  $\dot{\mathbf{x}}(t) = \mathbf{A}(t)\mathbf{x}(t)$ , defining  $\mathbf{H}(t) = \mathbf{A}(t) + \mathbf{A}^T(t)$ . Then (sufficient but not necessary):

1. State is bounded by:

$$\|\mathbf{x}(0)\| \frac{1}{2} \int_{t_0}^t \lambda_{\min}(\mathbf{H}(\tau)) d\tau \leq \|\mathbf{x}(t)\| \leq \|\mathbf{x}(0)\| \frac{1}{2} \int_{t_0}^t \lambda_{\max}(\mathbf{H}(\tau)) d\tau.$$

2. Stability:  $\lim_{t \rightarrow \infty} \int_{t_0}^t \lambda_{\min}(\mathbf{H}(\tau)) d\tau < M(t_0) < \infty$ , unifrom stability for  $M(t_0) = M$ .
3. Uniform asymptotic stability:  $\lim_{t \rightarrow \infty} \int_{t_0}^t \lambda_{\min}(\mathbf{H}(\tau)) d\tau = -\infty$ .
4. Instable:  $\lim_{t \rightarrow \infty} \int_{t_0}^t \lambda_{\min}(\mathbf{H}(\tau)) d\tau = +\infty$ .

**Lyoponovs Indirect Method:** The system is given as  $\dot{\mathbf{x}}(t) = \mathbf{A}(t)\mathbf{x}(t) + f_1(\mathbf{x}, t)$ , where  $\mathbf{A}(t) = \partial f / \partial \mathbf{x} |_{\mathbf{x}=\mathbf{0}}$ ,  $f_1(\mathbf{x}, t) = \mathbf{A}(t)\mathbf{x} - f(\mathbf{x}, t)$ , and

$$\lim_{\|\mathbf{x}\| \rightarrow 0} \sup_{t > 0} \frac{f_1(\mathbf{x}, t)}{\|\mathbf{x}\|} = 0.$$

This can be solved by performing a stability analysis for the linear system  $\dot{\mathbf{x}}(t) = \mathbf{A}(t)\mathbf{x}(t)$ . Asymptotic (Exponential/In-) stability of the linearized system implies local asymptotic (exponential/In-) stability of the nonlinear system. We can make use of this for finding stable feedback controllers.

**Determine Region of Attraction:** Determine (i) a compact set  $\Omega_e$  containing  $\mathbf{x}_e$  such that it is invariant under  $\dot{\mathbf{x}} = f(\mathbf{x})$  (for example  $\Omega_e = \{\mathbf{x} | V(\mathbf{x}) \leq c \vee \dot{V}(\mathbf{x}) < 0\}$ ), and (ii)  $\forall \mathbf{x} \neq 0 : \dot{V}(\mathbf{x}) < 0$  and  $V(\mathbf{0}) = 0$ . Follows from LaSalle-Krasovski.

## 5.4 Helpful Lyapunov Proofs

**Possible Lyapunov functions:** (i) use the total energy of the physical system, (ii) use  $V(\mathbf{x}) = \sum_i x_i^2/2 \implies \dot{V}(\mathbf{x}) = \sum_i x_i \dot{x}_i$ ,

**Mechanical Systems:** Given a mechanical system  $m\ddot{x} + f(\dot{x}) + g(x) = 0$ , with continuous  $f, g$ , so that  $\forall x.xf(x) \geq 0, \forall x.xg(x) \geq 0$ , we can use  $V(x) = x^2/2 + \int_0^x g(\sigma)d\sigma \implies \dot{V}(x) = -x_2 f(x_2) \leq 0$ . Stability follows through LaSalle-Krasovski.

**Exponential stability:** Use  $V(\mathbf{x}) = \mathbf{x}^T \mathbf{Q} \mathbf{x}$ , then we have  $\lambda_{\min}(\mathbf{Q}) \|\mathbf{x}\|^2 \leq V(\mathbf{x}, t) \leq \lambda_{\max}(\mathbf{Q}) \|\mathbf{x}\|^2$ , you only have to determine  $\dot{V}(\mathbf{x}, t) \leq -c \|\mathbf{x}\|^p$ .

**Instability:** Sometimes,  $V(\mathbf{x}) = x_1^2 - x_2^2$  is practical iff  $-\dot{V}(\mathbf{x}) = -x_1 \dot{x}_1 + x_2 \dot{x}_2 > 0$ .

## 5.5 Notes

- Stability is defined in terms of equilibrium points and NOT systems.
- Lyapunov stability notions are local.
- An EQ can only be stable or unstable.
- For an autonomous system, a stable EQ is also uniformly stable.
- Lyapunov functions are not unique.
- Interpretation: going downhill in direction  $\dot{\mathbf{x}}$  in a landscape  $V(\mathbf{x})$  within  $90^\circ$  of the steepest descent  $-\partial V(\mathbf{x})/\partial \mathbf{x}$ , i.e., at  $-\dot{V}(\mathbf{x}) = -(\partial V(\mathbf{x})/\partial \mathbf{x})^T \dot{\mathbf{x}} \geq 0$ .

## 5.6 Applications of Lyapunov Theory

**Feedback stabilization:** For a system  $\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u}, t), \mathbf{y} = g(\mathbf{x})$ , we intend to find  $\mathbf{u} = -p(\mathbf{y}, t)$  so that  $\lim_{t \rightarrow \infty} \mathbf{e}(t) = 0$  where  $\mathbf{e}(t) = \mathbf{y}(t) - \mathbf{y}_d$ . We can then simply stabilize the system  $\dot{\mathbf{x}} = f(\mathbf{x}, -p(g(\mathbf{x}), t), t)$ . *Example:* For mechanical systems  $\dot{\mathbf{x}} = f(\mathbf{x}, t) + M(\mathbf{x})\mathbf{u}$ , we can choose  $\mathbf{u} = -M^{-1}(\mathbf{x})[f(\mathbf{x}, t) - h(\mathbf{x})]$ , which turns the system into  $\dot{\mathbf{x}} = h(\mathbf{x})$  where  $h$  is chosen stable.

**Trajectory following:** When following a trajectory  $\dot{\mathbf{x}}_d = h(\mathbf{x}_d)$  with a system  $\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u}, t)$ , and a controller  $\mathbf{u} = p_1(\mathbf{x}_d - \mathbf{x}) + p_2(\dot{\mathbf{x}}_d)$ . We can then choose  $\xi = [\mathbf{x}_d, \mathbf{x} - \mathbf{x}_d] = [\mathbf{x}_d, \mathbf{e}]$  which yields  $\dot{\xi} = [h(\mathbf{x}_d), f(\mathbf{x}, \mathbf{u}, t)] =$

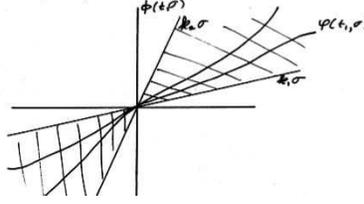


Figure 4: This figure shows the illustration for a sector.

$[h(\mathbf{x}_d), f(\mathbf{e} + \mathbf{x}_d, p_1(\mathbf{e}) + p_2(\mathbf{x}_d), t)]$ . We then choose the controller so that the origin is asymptotically stable.

**Adaptive control:** We have a system  $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{r}$ , and a (desired) model  $\dot{\mathbf{x}}_m = \mathbf{A}_m\mathbf{x}_m + \mathbf{B}_m\mathbf{r}_m$ . By defining  $\mathbf{e} = \mathbf{x}_m - \mathbf{x}$ , we obtain  $\dot{\mathbf{e}} = \mathbf{A}_m\mathbf{e} + (\mathbf{A}_m - \mathbf{A})\mathbf{x} + (\mathbf{B}_m - \mathbf{B})\mathbf{r}$ . Using the Lyoppanov function  $V(\mathbf{e}) = \mathbf{e}^T \mathbf{P}\mathbf{e} + \sum_{i=1}^n \sum_{j=1}^n \mu_{ij} \alpha_{ij}^2 + \sum_{i=1}^n \sum_{j=1}^m \nu_{ij} \beta_{ij}^2$  with  $\mu_{ij}, \nu_{ij} > 0$ ,  $[\alpha_{ij}] = (\mathbf{A}_m - \mathbf{A})$ ,  $[\beta_{ij}] = (\mathbf{B}_m - \mathbf{B})$ , and using the update rules

$$\dot{\alpha}_{ij} = -\frac{1}{\mu_{ij}} x_j \mathbf{e}^T \mathbf{p}_i, \quad \dot{\beta}_{ij} = -\frac{1}{\nu_{ij}} x_j \mathbf{e}^T \mathbf{p}_i$$

in  $\dot{V}$  (where  $\mathbf{P} = [\mathbf{p}_1, \dots, \mathbf{p}_2]$ ), we obtain  $\dot{V} = -\mathbf{e}^T \mathbf{e}$ . The error of the trajectory goes to zero but not the model parameters.

## 6 Frequency Domain Methods

**The Lure Problem:** The system is given by

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{b}u(t), \quad y(t) = \mathbf{c}^T \mathbf{x}(t) + du(t), \quad u(t) = -\varphi(y(t)).$$

The linearity is equivalent to  $G(s) = \mathbf{c}^T (s\mathbf{I} - \mathbf{A})^{-1} \mathbf{b} + d$ .

**Assumptions:** We assume that either  $\mathbf{A}$  is Hurwitz ( $\forall i. \text{Re } \lambda_i(\mathbf{A}) < 0$  or exactly one Eigenvalue is zero  $\lambda_1(\mathbf{A}) = 0, \forall i \geq 2. \text{Re } \lambda_i(\mathbf{A}) < 0$ ).  $(\mathbf{A}, \mathbf{b})$  controllable,  $(\mathbf{A}, \mathbf{c})$  observable. The nonlinearity is either odd ( $\varphi(0) = 0, \forall y \neq 0. y\varphi(y) > 0$ ), or in a sector (see below).

**$\varphi$  in a sector:**  $\varphi$  belongs to a sector  $[k_1, k_2]$ , ( $\varphi \in S(k_1, k_2)$ ), if

$$k_1 y^2 \leq y\varphi(y, t) \leq k_2 y^2.$$

Stability in a sector is called the absolute stability problem. See Figure 4 for an illustration.

**Aizerman's conjecture:** If  $d = 0, \varphi \in S(k_1, k_2), \forall k \in [k_1, k_2]. (\mathbf{A} - \mathbf{b}k\mathbf{c}^T)$  is Hurwitz, then the origin is globally asymptotically stable.

**Kalman's conjecture:** If  $\forall k \in [k_1, k_2]. (\mathbf{A} - \mathbf{b}k\mathbf{c}^T)$  is Hurwitz, and if

$$k_1 \leq \frac{\partial \varphi(y, t)}{\partial y} \leq k_2,$$

then the origin is globally asymptotically stable (stricter than Aizerman's conjecture).

**Kalman-Yakubovitch lemma:** If  $\mathbf{A}$  is Hurwitz,  $(\mathbf{A}, \mathbf{b})$  controllable,  $\mathbf{v} \in \mathbb{R}^n$ ,  $\gamma \geq 0$ ,  $\varepsilon > 0$ ,  $\mathbf{Q}$  pd, then there is pd  $\mathbf{P}$ , and  $\mathbf{q} \in \mathbb{R}^n$ , so that

$$\mathbf{A}^T \mathbf{P} + \mathbf{P} \mathbf{A} = -\mathbf{q}\mathbf{q}^T - \varepsilon \mathbf{Q}, \mathbf{P} \mathbf{b} - \mathbf{v} = \sqrt{\gamma} \mathbf{q},$$

if and only if  $\varepsilon$  small, and  $h(s) = \gamma + 2\mathbf{v}^T (s\mathbf{I} - \mathbf{A})^{-1} \mathbf{b}$  satisfies  $\forall \omega \in \mathbb{R}. \operatorname{Re} h(j\omega) > 0$ .

**Circle criterion:** If  $\varphi \in S(\alpha, \beta)$ , the origin of the system will be absolutely stable/globally asymptotically stable if one of the following four sufficient graphical conditions applies (these are illustrated in Table 2).

1. If  $0 < \alpha < \beta$ , the nyquist plot has to encircle the system  $m$  times in counterclockwise direction, where  $m$  is the number of poles with positive real part.
2. If  $0 = \alpha < \beta$ , the nyquist plot lies above  $-1/\beta$ .
3. If  $\alpha < 0 < \beta$ , the nyquist plot lies in the interior of the disk  $D(\alpha, \beta)$ .
4. If  $\alpha < \beta < 0$ , then use  $\hat{g} = -g$ ,  $\hat{\alpha} = -\beta$ ,  $\hat{\beta} = -\alpha$ , and apply (1).

Alternatively, one could interpret it analytically as  $\phi \in S(0, k)$ ,  $1 + kd > 0$ ,  $\operatorname{Re}\{1 + kg(j\omega)\} > 0 \implies$  absolute stability.

**Popov criterion:** The system is slightly modified to

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{b}u, y = \mathbf{c}^T \mathbf{x} + d\xi, \dot{\xi} = u, u = -\varphi(y),$$

i.e., with a transfer function  $G(s) = \mathbf{c}^T (s\mathbf{I} - \mathbf{A})^{-1} \mathbf{b} + d/s$ , which has to be Hurwitz, or one Eigenvalue that is zero. The system is globally asymptotically stable if there exists a number  $r$  such that  $\operatorname{Re}\{1 + j\omega r g(j\omega)\} + 1/k > 0$ , or equivalently

$$\operatorname{Re}\{g^*(j\omega)\} > -\frac{1}{k} + r \operatorname{Im} g^*(j\omega),$$

with  $g^*(j\omega) = \operatorname{Re} g(j\omega) + j\omega \operatorname{Im} g(j\omega)$ . The graphical interpretation is given in Figure 5.

	Disk $D(\alpha, \beta)$	Hurwitz?	Sketch
Case (1)	$0 < \alpha < \beta$	-	
Case (2)	$0 = \alpha < \beta$	Required	
Case (3)	$\alpha < 0 < \beta$	Required	
Case (4)	$\alpha < \beta < 0$	-	

Table 2: This table shows the four cases of the circle criterion.

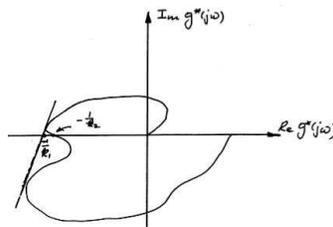


Figure 5: This figure shows the Popov criterion.

## 7 Extensions of the Notion of Stability

### 7.1 Boundedness

**Uniformly bounded:**  $\forall a \in (0, c). \exists b(a). \forall t \geq t_0. \|x(t_0)\| \leq a \implies \|x(t)\| < b.$

**Globally uniformly bounded:**  $\forall a \in \mathbb{R}. \exists b(a). \forall t \geq t_0. \|x(t_0)\| \leq a \implies \|x(t)\| < b.$

**Uniformly ultimately bounded:**  $\forall a \in \mathbb{R}. \exists b(a). \forall t \geq t_0 + T(a, b). \|x(t_0)\| \leq a \implies \|x(t)\| < b.$

*To be continued...*

## 7.2 Perturbations

**Additive perturbation:** We assume that the system can be split into a nominal model  $f$  and an additive perturbation  $g$ , so that

$$\dot{\mathbf{x}} = f(\mathbf{x}, t) + g(\mathbf{x}, t) = f(\mathbf{x}, t) + (f(\mathbf{x}, t) - \bar{f}(\mathbf{x}, t)).$$

Any same/lower order perturbation can be represented like this.

**Vanishing perturbations:** The origin is an equilibrium point of the perturbation, i.e.,  $g(\mathbf{x}, t) = 0$ .

**Robustness of exponential stability:** Assume  $\mathbf{x} = 0$  an exponentially stable equilibrium point of  $f$ , if  $f$  is exponentially stable on  $D$ , i.e.,

$$\forall \mathbf{x} \in D. c_1 \|\mathbf{x}\|^2 \leq V(\mathbf{x}, t) \leq c_2 \|\mathbf{x}\|^2, c_3 \|\mathbf{x}\|^2 \leq - \left( \frac{\partial V}{\partial t} + \frac{\partial V}{\partial \mathbf{x}} f(\mathbf{x}, t) \right),$$

and  $\forall \mathbf{x} \in D. \|\partial V / \partial \mathbf{x}\| < c_4 \|\mathbf{x}\|^2$ . If furthermore, the perturbation fulfills  $\forall \mathbf{x} \in D. \|g(\mathbf{x}, t)\| < \gamma \|\mathbf{x}\|$  with  $\gamma < c_3/c_4$ , the system is exponentially stable. If  $D = \mathbb{R}^n$ , even globally.

**Linear Systems Nonlinearly Perturbed:** Assume  $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + g(\mathbf{x}, t)$ ,  $\text{Re}\{\lambda_i(\mathbf{A})\} < 0$ , and  $\|g(\mathbf{x}, t)\| < \gamma \|\mathbf{x}\|$ . If  $\mathbf{P}$  is a solution of the Lyapunov equation  $\mathbf{P}\mathbf{A} + \mathbf{A}^T\mathbf{P} = -\mathbf{Q}$ , and  $\gamma < \lambda_{\min}(\mathbf{Q})/(2\lambda_{\max}(\mathbf{P}))$ , the system will be globally exponentially stable. This ratio  $\gamma$  is maximal for  $\mathbf{Q} = \mathbf{I}$ .

**Robustness of Asymptotic Stability:** Assume  $\mathbf{x} = 0$  an asymptotically stable equilibrium point of  $f$ , i.e.,

$$W_1(\mathbf{x}) \leq V(\mathbf{x}, t) \leq W_2(\mathbf{x}), c_3 W_{3/4}^2(\mathbf{x}) \leq - \left( \frac{\partial V}{\partial t} + \frac{\partial V}{\partial \mathbf{x}} f(\mathbf{x}, t) \right),$$

and  $\|\partial V / \partial \mathbf{x}\| < c_4 W_{3/4}(\mathbf{x})$ ; all  $W_i(\mathbf{x})$  are pdf. If furthermore, the perturbation fulfills  $\|g(\mathbf{x}, t)\| < \gamma W_{3/4}(\mathbf{x})$  with  $\gamma < c_3/c_4$ , the system is asymptotically stable.

**Nonvanishing Perturbations:** Assume  $\mathbf{x} = 0$  an exponentially stable equilibrium point of  $f$ , if  $f$  is exponentially stable on  $D = \{\mathbf{x} \in \mathbb{R}^n \mid \|\mathbf{x}\| < r\}$ , i.e.,

$$\forall \mathbf{x} \in D. c_1 \|\mathbf{x}\|^2 \leq V(\mathbf{x}, t) \leq c_2 \|\mathbf{x}\|^2, c_3 \|\mathbf{x}\|^2 \leq - \left( \frac{\partial V}{\partial t} + \frac{\partial V}{\partial \mathbf{x}} f(\mathbf{x}, t) \right),$$

and  $\forall \mathbf{x} \in D. \|\partial V / \partial \mathbf{x}\| < c_4 \|\mathbf{x}\|^2$ . If furthermore, the perturbation fulfills

$$\forall \mathbf{x} \in D. \|g(\mathbf{x}, t)\| \leq \delta < \frac{c_3}{c_4} \sqrt{\frac{c_1}{c_2}} \theta r,$$

for some  $0 < \theta < 1$ . Then for all  $\|\mathbf{x}(0)\| < r\sqrt{c_1/c_2}$ ,

$$\forall t_0 \leq t \leq T. \|\mathbf{x}(t)\| \leq k \exp[-\gamma(t - t_0)] \|\mathbf{x}(0)\| \wedge \|\mathbf{x}(t)\| \leq b,$$

for finite  $T$ , where  $k = \sqrt{c_2/c_1}$ ,  $\gamma = (1-\theta)c_3/(2c_2)$ ,  $b = (c_3/c_4)\sqrt{c_1/c_2}(\delta/\theta)$ .

## 8 Nonlinear Control Design

### 8.1 Introduction to Feedback Control Design

**Objectives of Control Design:** Equilibrium point stabilization, Trajectory tracking, Disturbance rejection (input/output boundedness), robustness (cope with model errors).

**State feedback stabilization:** Show that the desired equilibrium point in

$$\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u}, t), \mathbf{u} = \gamma(\mathbf{x}, \mathbf{z}, t), \dot{\mathbf{z}} = f(\mathbf{x}, \mathbf{z}, t),$$

is asymptotically stable. Use  $\mathbf{z}$  in order to implement I-controllers.

**Output feedback stabilization:** Show that the desired equilibrium point in

$$\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u}, t), \mathbf{y} = h(\mathbf{x}, \mathbf{u}, t), \mathbf{u} = \gamma(\mathbf{y}, \mathbf{z}, t), \dot{\mathbf{z}} = f(\mathbf{y}, \mathbf{z}, t),$$

is asymptotically stable. Use  $\mathbf{z}$  in order to implement I-controllers. For moving the equilibrium point, all variables  $\hat{\mathbf{x}} = \mathbf{x} - \mathbf{x}_0$ ,  $\hat{\mathbf{u}} = \mathbf{u} - \mathbf{u}_0$ ,  $\hat{\mathbf{y}} = h(\mathbf{x}_0 + \hat{\mathbf{x}}, \mathbf{u}_0 + \hat{\mathbf{u}}, t) - h(\mathbf{x}_0, \mathbf{u}_0, t)$ , have to be moved.

**Linear Systems:** For linear time-invariant systems ( $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u}$ ,  $\mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{u}$ ,  $\mathbf{u} = -\mathbf{K}\mathbf{x}$ ), both problems become easier. State feedback linearization requires only  $\text{Re } \lambda_i(\mathbf{A} - \mathbf{B}\mathbf{K}) < 0$ . Output stabilization requires an Observer  $d\hat{\mathbf{x}}/dt = (\mathbf{A} - \mathbf{B}\mathbf{K})\hat{\mathbf{x}} + \mathbf{L}(\mathbf{x} - \mathbf{C}\hat{\mathbf{x}} - \mathbf{D}\mathbf{u})$ , so that we analyze

$$\begin{bmatrix} \dot{\hat{\mathbf{x}}} \\ \dot{\mathbf{e}} \end{bmatrix} = \begin{bmatrix} \mathbf{A} - \mathbf{B}\mathbf{K} & \mathbf{B}\mathbf{K} \\ \mathbf{0} & \mathbf{A} - \mathbf{L}\mathbf{C} \end{bmatrix} \begin{bmatrix} \hat{\mathbf{x}} \\ \mathbf{e} \end{bmatrix}.$$

where  $\mathbf{e} = \mathbf{x} - \hat{\mathbf{x}}$  represents the error.

**Stabilization by Linearization:** We can stabilize an equilibrium point of the nonlinear system by linearization. We can determine the region of stability using a quadratic Lyapunov function.

### 8.2 Closed-loop Control

*To be continued...*

### 8.3 Integral Control / Gain Scheduling

**System description:** The system is described by

$$\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u}, \mathbf{w}), \dot{\sigma} = h(\mathbf{x}, \mathbf{w}) - \mathbf{r},$$

where  $\mathbf{w}$  is unknown.

**Stabilizing Control Law:** A stabilizing control law  $\mathbf{u} = \gamma(\mathbf{x}, \sigma, h(\mathbf{x}, \mathbf{w}) - \mathbf{r})$  has to fulfill (i)  $\mathbf{u}_0 = \gamma(\mathbf{x}_0, \sigma_0, 0)$ , and (ii) that  $\dot{\mathbf{x}} = f(\mathbf{x}, \gamma(\mathbf{x}, \sigma, h(\mathbf{x}, \mathbf{w}) - \mathbf{r}), \mathbf{w})$ ,  $\dot{\sigma} = h(\mathbf{x}, \mathbf{w}) - \mathbf{r}$ , has to have an asymptotically stable equilibrium point at  $(\mathbf{x}_0, \mathbf{u}_0)$ .

**Via Linearization:** Use the linear control law  $\mathbf{u} = -\mathbf{K}_1\mathbf{x} - \mathbf{K}_2\sigma - \mathbf{K}_3\mathbf{e}$  with error  $\mathbf{e} = h(\mathbf{x}, \mathbf{w}) - \mathbf{r}$ . If  $\mathbf{K}_2$  is nonsingular, there is a unique solution for  $\sigma_0$  so that  $\mathbf{u}_0 = -\mathbf{K}_1\mathbf{x}_0 - \mathbf{K}_2\sigma_0$ .

**Gain scheduling:** For tracking a trajectory  $r(t)$ , we treat each point of the trajectory as an equilibrium point, i.e., we have gains depending on  $r$  so that  $\mathbf{u} = -\mathbf{K}_1(r)\mathbf{x} - \mathbf{K}_2(r)\sigma - \mathbf{K}_3(r)\mathbf{e}$  is stable if  $r$  was constant.

### 8.4 Sliding Mode Control

**Objective:** Assume a system

$$\begin{aligned} \dot{x}_1 &= x_2, \\ \dot{x}_2 &= h(x_1, x_2) + g(x_1, x_2)u, \end{aligned}$$

where  $g, h$  are unknown and  $g(x_1, x_2) \geq g(x_{10}, x_{20}) > 0$ . We intend bring the system to the manifold  $s = a_1x_1 + x_2 = 0$  with arbitrary  $a_1$ .

**Control Law:** Assume that you know

$$\left\| \frac{a_1x_2 + h(x_1, x_2)}{g(x_1, x_2)} \right\| \leq \rho(x_1, x_2),$$

and the Lyapanov function  $V(s) = 0.5s^2$ , and  $\dot{V}(s) = s\dot{s} \leq g(x_1, x_2)|s|\rho(x_1, x_2) + s \cdot g(x_1, x_2)u$ . Then

$$u = -\beta(x_1, x_2) \text{sign } s,$$

is a stable control law if  $\beta(x_1, x_2) > \rho(x_1, x_2) + \beta_0$  for some  $\beta_0 > 0$ . Nota bene: (i) Manifold is reached in finite time, and  $x_2 = \dot{x}_1 = -ax_1$  converges exponentially fast. (ii) Once reached, the manifold cannot be left. (iii) Robust to parameter variation.

**Chattering:** Delays in system or controller can cause chattering.

**Chattering reduction by Separation:** Separate the continuous and switching components. Use nominal  $\hat{h}, \hat{g}$  in the control law

$$u = - \left\| \frac{a_1 x_2 + \hat{h}(x_1, x_2)}{\hat{g}(x_1, x_2)} \right\| - \beta(x_1, x_2) \text{sign } s,$$

where  $\beta(x_1, x_2) > \rho(x_1, x_2) + \beta_0$  for some  $\beta_0 > 0$ , if

$$\left\| \frac{\delta(x_1, x_2)}{g(x_1, x_2)} \right\| \leq \rho(x_1, x_2)$$

with  $\delta(x_1, x_2) = a_1(1-g(x_1, x_2)/\hat{g}(x_1, x_2))x_2 + (h(x_1, x_2) - \hat{h}(x_1, x_2)g(x_1, x_2)/\hat{g}(x_1, x_2))$ .

**Chattering reduction of high gain saturation function:** Use the control law with saturation

$$u = -\beta(x_1, x_2) \text{sat} \left( \frac{s}{\varepsilon} \right),$$

and a Lyapunov function  $V = 0.5x_1^2$ , we can show ultimate boundedness so that all trajectories reach the (invariant) set

$$\Omega_\varepsilon = \left\{ x_1 \leq \frac{\varepsilon}{a_1}, s \leq \varepsilon \right\},$$

in finite time. If  $\varepsilon$  decreases, the ultimate bound  $T$  and chattering increases  $\rightarrow$  trade-off stability vs performance.

**Generalization:** We can generalize using the system  $y^{(n)} = h(\mathbf{x}) + g(\mathbf{x})u$  with  $\mathbf{x} = [y, \dot{y}, \dots, y^{(n-1)}]$ . Assume bounds  $\|h(\mathbf{x})\| < \varepsilon(\mathbf{x})$ ,  $\|g(\mathbf{x})\| < \mu(\mathbf{x})$ . We use this for tracking a trajectory  $\mathbf{x}_d(t)$ , i.e., we have the manifold

$$s(\mathbf{x}) = e_y^{(n-1)} + a_1 e_y^{(n-2)} + \dots + a_{n-1} e_y = 0$$

using  $e = y - y_d$ . For  $s(\mathbf{x}) = (d/dt + \lambda)^{n-1} e_y$ , we can determine whether the  $a_i$ 's are suitable by checking  $z^{n-1} + a_1 z^{n-2} + \dots + a_{n-1} = 0$  is Hurwitz.

## 8.5 Lyapunov Redesign

**System:** The system with perturbation  $\delta$  is given by  $\dot{x} = f(x, u, t) + g(x, t)[u + \delta(x, u, t)]$ . Assume, you have a controller  $u = \psi(x, t)$ , which stabilizes the nominal system (i.e.,  $\delta(x, u, t) = 0$ ).

**Lyapunov function:** Determine a Lyapunov function so that

$$\alpha_1 (\|\mathbf{x}\|) \leq V(\mathbf{x}, t) \leq \alpha_2 (\|\mathbf{x}\|), \frac{\partial V}{\partial t} + \frac{\partial V}{\partial \mathbf{x}} [f(\mathbf{x}, t) + g(x, t)\psi(x, t)] \leq -\alpha_3 (\|\mathbf{x}\|).$$

**Perturbation Assumption:** Assume that for  $u = \psi(x, t) + v$ , we have the bound

$$\|\delta(x, \psi(x, t) + v, t)\| \leq \rho(x, t) + \varkappa_0 \|v\|,$$

with  $0 < \varkappa_0 < 1$ . Nota bene: Only bounds on perturbation,  $\rho(x, t)$  can be large but has to be known.

**Design of  $v$ :** Using  $\dot{V} \leq -\alpha_3 (\|x\|) + w^T v + w^T \delta$  with  $w = (\partial V / \partial x)g(x, t)$ , we realize that  $w^T v + w^T \delta \leq 0$ .

1. **Norm 2:** This is achieved by

$$v = -\eta(x, t) \frac{w}{\|w\|}$$

with  $\eta(x, t) \geq \rho(x, t) / (1 - \varkappa_0)$ . This gives us the control law

$$u = \psi(x, t) - \frac{\rho(x, t)}{1 - \varkappa_0} \frac{w}{\|w\|},$$

which is stable.

2. **Norm  $\infty$ :** This is achieved by

$$v = -\eta(x, t) \operatorname{sign} w$$

with  $\eta(x, t) \geq \rho(x, t) / (1 - \varkappa_0)$ .

3. **Norm 1:** This is achieved by

$$v = -\eta(x, t) \frac{w}{\|w\|}$$

with  $\eta(x, t) = \rho(x, t) / (1 - \varkappa_0)$ .

## 8.6 Backstepping

*To be continued...*